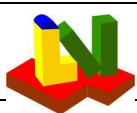


## Table of contents

Table of contents .....	1
Summary .....	2
E4.3.7 Large End - Cone-to-cylinder junction at large end under internal pressure - ASME BPVC VIII-1 UG-32 & Appendix-1: 2017 .....	3
E4.3.7 Small End - Cone-to-cylinder junction at small end under internal pressure - ASME BPVC VIII-1 UG-32 & Appendix-1: 2017 .....	4
E4.3.8 - Cone with knuckle under internal pressure - ASME BPVC VIII-1 UG-32 & Appendix-1: 2017 .....	6
Appendix: Material documentation .....	8

## Layout

Input values:	1.234	or	1.234
Calculated values:	<b>1.234</b>	or	<b>1.234</b>
Critical values:	<b>1.234</b>	or	<b>1.234</b>
Estimated values:	1.234	or	1.234

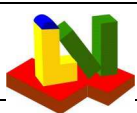


**ASME BPVC VIII-1 2017**  
**Example E4.3.7 - E4.3.8 PTB-4-2013**

**Summary**

Strength Calculation Software	Program System ATLAS	Version	<b>8.29.1</b>
Developed by Lauterbach Verfahrenstechnik GmbH			
Certified per DIN EN ISO	Certificate Number		
9001:2015	01 100 044763		

	LV Soft	ASME	Diff [%]
Example E4.3.7 - Conical transition Without a knuckle (large End)			
Req. cylinder thickness t	37,48 mm 1,48 in	1,48 in	<b>0,08%</b>
Req. cone thickness tr	39,93 mm 1,57 in	1,57 in	<b>0,08%</b>
Example E4.3.7 - Conical transition Without a knuckle (Smal End)			
Req. cylinder thickness t	23,78 mm 0,94 in	0,94 in	<b>0,07%</b>
Req. cone thickness tr	25,25 mm 0,99 in	0,99 in	<b>0,07%</b>
Required area Ars	2.087,34 mm <sup>2</sup> 3,24 in <sup>2</sup>	3,24 in <sup>2</sup>	<b>0,03%</b>
Required area Aes	4.054,38 mm <sup>2</sup> 6,28 in <sup>2</sup>	6,28 in <sup>2</sup>	<b>0,11%</b>
Example E4.3.8 - Conical transition With a knuckle			
Req. knuckle thickness t	17,20 mm 0,68 in	0,68 in	<b>0,07%</b>
Req. cone thickness tr	24,74 mm 0,97 in	0,97 in	<b>0,08%</b>



### E4.3.7 Large End - Cone-to-cylinder junction at large end under internal pressure - ASME BPVC VIII-1 UG-32 & Appendix-1: 2017

#### Cone-to-cylinder junction (wide end)

Type of reinforcing ring

(0 = no, 1 = at the cylinder, 2 = at the cone)

**Without stiffener**

Design pressure

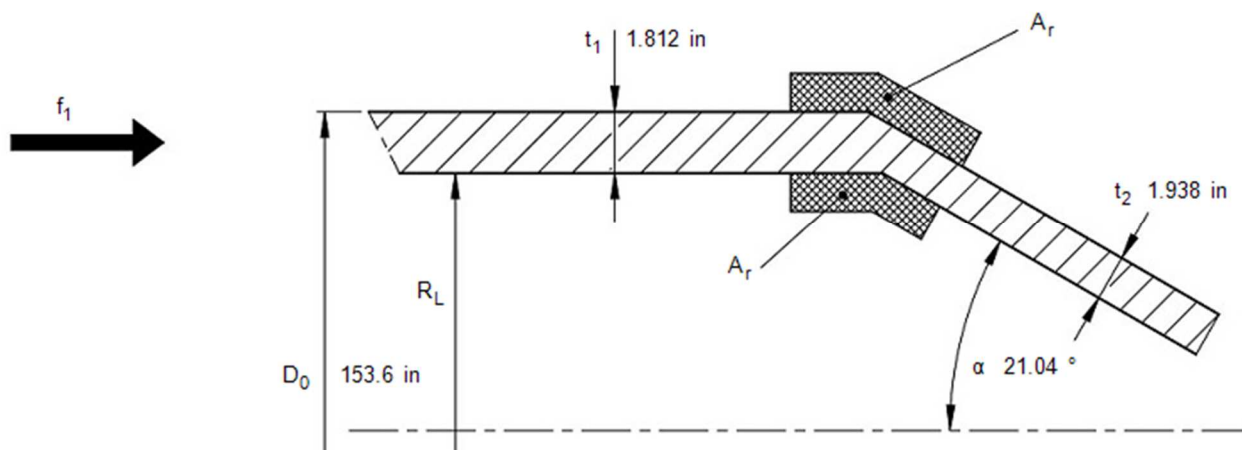
Hydrostatic head

Calculation pressure

Calculation temperature

Axial load based on circumference (for compression negative)

$p_D$	24.55 bar
$D_p$	0 bar
$p_0$	<b>356</b> psi
$T_0$	148.9 °C
$f_1$	0 lbf/in



#### Cylinder

Outside diameter

Final wall thickness

Material K02700-SA-516-70-Class:-Size:

Wall thickness allowance

Allowance (corrosion)

Thickness without allowances

Inside radius

(=  $D_0/2 - t_s$ )

Allowable stress

Modulus of elasticity

Joint efficiency factor

$D_0$	153.6 in
$t_1$	1.812 in
$c_1$	0 in
$c_2$	0.125 in
$t_s$	<b>1.687</b> in
$R_L$	<b>75.12</b> in
$S_s$	20015 psi
$E_s$	2.829e+7 psi
$E_1$	1

#### Cone

Half-apex angle ( $\leq 30^\circ$ )

Final wall thickness

Material K02700-SA-516-70-Class:-Size:

Wall thickness allowance

Allowance (corrosion)

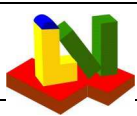
Effective thickness

Allowable stress

Modulus of elasticity

Joint efficiency factor

$\alpha$	21.04 °
$t_2$	1.938 in
$c_1$	0 in
$c_2$	0.125 in
$t_c$	<b>1.813</b> in
$S_c$	20015 psi
$E_c$	2.829e+7 psi
$E_2$	1



## Results

Geometrical conditions

**valid**

Strength condition

**Wall thickness acceptable**

Factor			k	<b>1</b>
Ratio		$P_0/S_s E_1$		<b>0.01779</b>
Angle	(Reinforcement required if $\Delta < \alpha$ )	$21.04^\circ$	$\Delta$	<b>30°</b>
Effective load			$Q_L$	<b>13371</b> lbf/in
Required thickness cylinder (UG-27)			t	<b>1.351</b> in
with allowances ( $t_1$ )	1.812 in	$\geq t_+$	$t_+$	<b>1.476</b> in
Required thickness cone (UG-32)			$t_r$	<b>1.447</b> in
with allowances ( $t_2$ )	1.938 in	$\geq t_+$	$t_r +$	<b>1.572</b> in
Required cross sectional area			$A_{rL}$	<b>0</b> in <sup>2</sup>
Available cross section			$A_{eL}$	<b>8.209</b> in <sup>2</sup>
Required area of reinforcement			$A_r$	<b>0</b> in <sup>2</sup>
Available area of reinforcement		$b_v \cdot t_v$	$A_v$	<b>0</b> in <sup>2</sup>
Maximum distance from the connection point of the complete reinforcing area		$\sqrt{[R_L \cdot t_s]}$		<b>11.26</b> in
centroid of reinforcing area		$0.25 \cdot \sqrt{[R_L \cdot t_s]}$		<b>2.815</b> in

## Equations

$$\cos(\alpha) = \cos(\alpha) \Leftrightarrow \cos(21.04^\circ) = 0.9333$$

$$\tan(\alpha) = \tan(\alpha) \Leftrightarrow \tan(21.04^\circ) = 0.3846$$

$$Q_L = P_0 \cdot \frac{R_L}{2} + f_1 = 24.55 \text{ bar} \cdot \frac{1908 \text{ mm}}{2} + 0 \text{ N/mm} = 2342 \text{ N/mm}$$

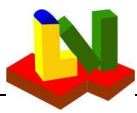
$$A_{rL} = \frac{k \cdot Q_L \cdot R_L}{S_s \cdot E_1} \cdot \left[ 1 - \frac{D_{el}}{\alpha} \right] \cdot \tan(\alpha) = \frac{1 \cdot 2342 \text{ N/mm} \cdot 1908 \text{ mm}}{138 \text{ N/mm}^2 \cdot 1} \cdot \left[ 1 - \frac{30^\circ}{21.04^\circ} \right] \cdot 0.3846 = 0 \text{ mm}^2$$

$$t = \frac{P_0 \cdot R_L}{S_s \cdot E_1 - 0.6 \cdot P_0} = \frac{24.55 \text{ bar} \cdot 1908 \text{ mm}}{138 \text{ N/mm}^2 \cdot 1 - 0.6 \cdot 24.55 \text{ bar}} = 34.3 \text{ mm}$$

$$t_r = \frac{P_0 \cdot \frac{R_L}{\cos(\alpha)}}{S_c \cdot E_2 - 0.6 \cdot P_0} = \frac{24.55 \text{ bar} \cdot \frac{1908 \text{ mm}}{0.9333}}{138 \text{ N/mm}^2 \cdot 1 - 0.6 \cdot 24.55 \text{ bar}} = 36.75 \text{ mm}$$

$$A_{eL} = (t_s - t) \cdot \sqrt{R_L \cdot t_s} + (t_c - t_r) \cdot \sqrt{R_L \cdot \frac{t_c}{\cos(\alpha)}} =$$

$$(42.86 \text{ mm} - 34.3 \text{ mm}) \cdot \sqrt{1908 \text{ mm} \cdot 42.86 \text{ mm}} + (46.04 \text{ mm} - 36.75 \text{ mm}) \cdot \sqrt{1908 \text{ mm} \cdot \frac{46.04 \text{ mm}}{0.9333}} = 5296 \text{ mm}^2$$



### E4.3.7 Small End - Cone-to-cylinder junction at small end under internal pressure - ASME BPVC VIII-1 UG-32 & Appendix-1: 2017

#### Cone-to-cylinder junction (small end)

Type of reinforcing ring

(0 = no, 1 = at the cylinder, 2 = at the cone)

0

**Without stiffener**

Design pressure

$p_D$  24.55 bar

Hydrostatic head

$D_p$  0 bar

Calculation pressure

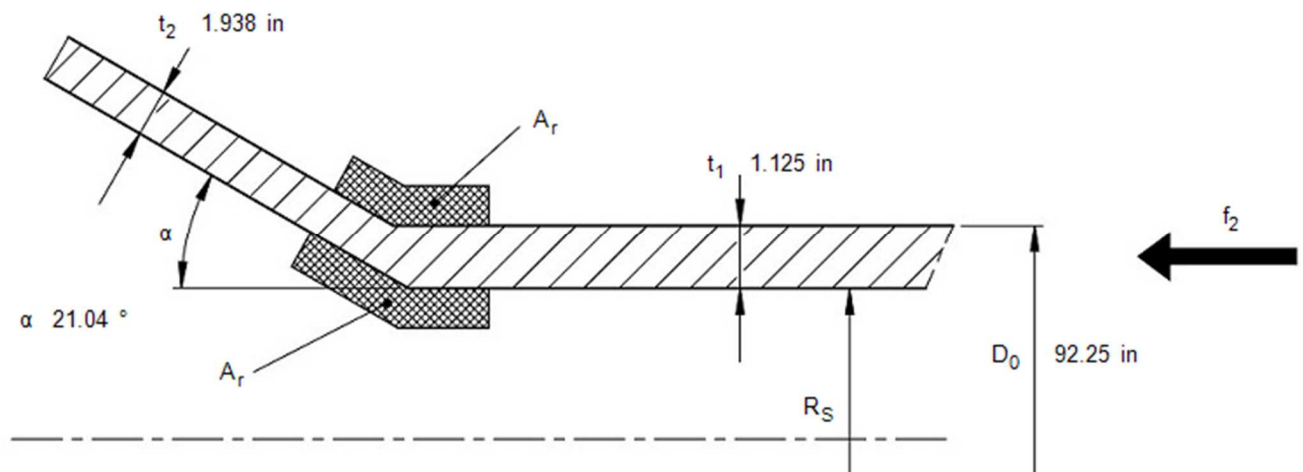
$p_0$  **356** psi

Calculation temperature

$T_0$  148.9 °C

Axial load based on circumference (for compression negative)

$f_2$  396.9 lbf/in



#### Cylinder

Outside diameter

$D_0$  92.25 in

Final wall thickness

$t_1$  1.125 in

Material K02700-SA-516-70-Class:-Size:

Wall thickness allowance

$c_1$  0 in

Allowance (corrosion)

$c_2$  **0.125** in

Effective thickness

$t_s$  **1** in

Inside radius

(=  $D_0/2 - t_s$ )

$R_S$  **45.12** in

Allowable stress

$S_s$  20015 psi

Modulus of elasticity

$E_s$  2.829e+7 psi

Joint efficiency factor

$E_1$  1

#### Cone

Half-apex angle ( $\leq 30^\circ$ )

$\alpha$  21.04 °

Final wall thickness

$t_2$  1.938 in

Material K02700-SA-516-70-Class:-Size:

Wall thickness allowance

$c_1$  0 in

Allowance (corrosion)

$c_2$  **0.125** in

Thickness without allowances

$t_c$  **1.813** in

Allowable stress

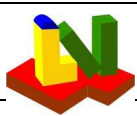
$S_c$  20015 psi

Modulus of elasticity

$E_c$  2.829e+7 psi

Joint efficiency factor

$E_2$  1



## Results

Geometrical conditions

**valid**

Strength condition

**Wall thickness acceptable**

Factor		k	<b>1</b>
Ratio	$P_0/S_s E_1$		<b>0.01779</b>
Angle		$\Delta$	<b>11.73</b> °
Effective load		$Q_s$	<b>1476</b> N/mm
Required thickness cylinder (UG-27)		t	<b>0.8113</b> in
with allowances ( $t_1$ )	1.125 in $\geq t_+$	$t_+$	<b>0.9363</b> in
Required thickness cone (UG-32)		$t_r$	<b>0.8692</b> in
with allowances ( $t_2$ )	1.938 in $\geq t_r$	$t_r$	<b>0.9942</b> in
Required cross sectional area		$A_{rS}$	<b>3.235</b> in <sup>2</sup>
Available cross section		$A_{eS}$	<b>6.284</b> in <sup>2</sup>
Required area of reinforcement		$A_r$	<b>0</b> in <sup>2</sup>
Available area of reinforcement	$b_v \cdot t_v$	$A_v$	<b>0</b> in <sup>2</sup>
Maximum distance from the connection point of the complete reinforcing area	$\sqrt{[R_L \cdot t_s]}$		<b>9.044</b> in
centroid of reinforcing area	$0.25 \cdot \sqrt{[R_L \cdot t_s]}$		<b>2.261</b> in

## Equations

$$\cos(\alpha) = \cos(\alpha) \Leftrightarrow \cos(21.04^\circ) = 0.9333$$

$$\tan(\alpha) = \tan(\alpha) \Leftrightarrow \tan(21.04^\circ) = 0.3846$$

$$Q_s = P_0 \cdot \frac{RS}{2} + f_2 = 24.55 \text{ bar} \cdot \frac{1146 \text{ mm}}{2} + 69.5 \text{ N/mm} = 1476 \text{ N/mm}$$

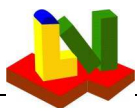
$$A_{rS} = \frac{k \cdot Q_s \cdot RS}{S_s \cdot E_1} \cdot \left[ 1 - \frac{Del}{\alpha} \right] \cdot \tan(\alpha) = \frac{1 \cdot 1476 \text{ N/mm} \cdot 1146 \text{ mm}}{138 \text{ N/mm}^2 \cdot 1} \cdot \left[ 1 - \frac{11.73^\circ}{21.04^\circ} \right] \cdot 0.3846 = 2087 \text{ mm}^2$$

$$t = \frac{P_0 \cdot RS}{S_s \cdot E_1 - 0.6 \cdot P_0} = \frac{24.55 \text{ bar} \cdot 1146 \text{ mm}}{138 \text{ N/mm}^2 \cdot 1 - 0.6 \cdot 24.55 \text{ bar}} = 20.61 \text{ mm}$$

$$t_r = \frac{P_0 \cdot \frac{RS}{(\cos(\alpha))}}{S_c \cdot E_2 - 0.6 \cdot P_0} = \frac{24.55 \text{ bar} \cdot \frac{1146 \text{ mm}}{0.9333}}{138 \text{ N/mm}^2 \cdot 1 - 0.6 \cdot 24.55 \text{ bar}} = 22.08 \text{ mm}$$

$$A_{eS} = 0.78 \cdot \sqrt{RS \cdot t_s} \cdot \left[ (ts - t) + \frac{(tc - tr)}{\cos(\alpha)} \right] =$$

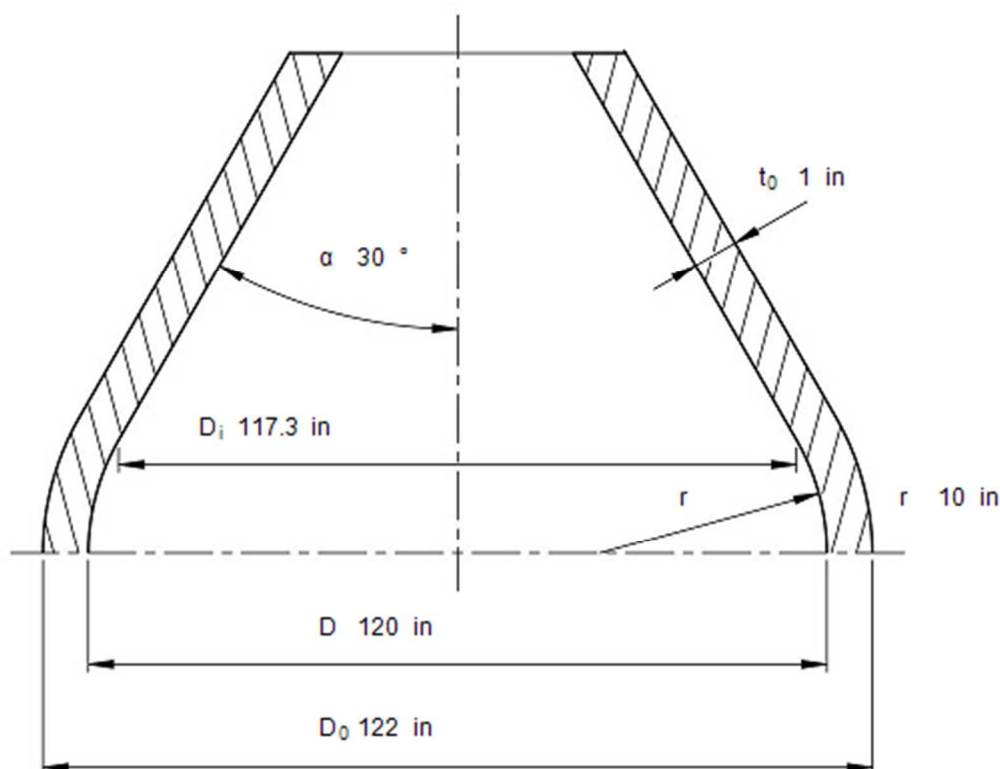
$$0.78 \cdot \sqrt{1146 \text{ mm} \cdot 25.4 \text{ mm}} \cdot \left[ (25.4 \text{ mm} - 20.61 \text{ mm}) + \frac{(46.04 \text{ mm} - 22.08 \text{ mm})}{0.9333} \right] = 4054 \text{ mm}^2$$



**E4.3.8 - Cone with knuckle under internal pressure - ASME BPVC VIII-1 UG-32 & Appendix-1: 2017**

**Toriconical sections**

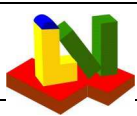
Design pressure	$p_D$	19.68 bar
Hydrostatic head	$D_p$	0 bar
Calculation pressure	$p_0$	<b>285.4</b> psi
Calculation temperature	$T_0$	148.9 °C
Final wall thickness	$t_e$	1 in
Wall thickness allowance	$c_1$	0 in
Allowance (corrosion)	$c_2$	0 in
Effective thickness without allowances	$t_0$	<b>1</b> in



Outside diameter of cylindrical shell	$D_0$	122 in
Inside diameter of cylindrical shell	$D$	<b>120</b> in
Semi-apex angle	$\alpha$	30 °
Knuckle radius ( $\geq 0.06 \cdot D_0$ , $\geq 3 \cdot t_0$ )	$r$	10 in
Weld joint efficiency factor	$E$	1

Material K02700-SA-516-70-Class:-Size:

Allowable stress S **20015** psi



# ASME BPVC VIII-1 2017

## Example E4.3.7 - E4.3.8 PTB-4-2013

### Calculation

Largest inside diameter of cone	$D_i$	117.3 in
Equivalent radius	$L$	67.74 in
Ratio	$L/r$	6.774
Factor	$M$	1.401
Required knuckle thickness	$t$	0.6773 in
Allowable inside pressure of knuckle	$P$	421 psi

Calculation diameter of cone	$D_1$	117.3 in
Required cone thickness at $D_1$	$t_1$	0.9741 in
Allowable inside pressure of cone	$P_1$	292.9 psi

### Remark

Required thickness incl. allowances	$t+c_1+c_2$	$t+$	0.9741 in
Allowable excess pressure	$\text{Min}(P, P_1)$	$P_m$	292.9 psi
Allowable excess pressure without hydr. Head		MAWP	292.9 psi

### Geometrical conditions

valid

### Strength condition

Wall thickness acceptable

### Equations knuckle

$$\cos(\alpha) = \cos(\alpha) \Leftrightarrow \cos(30^\circ) = 0.866$$

$$D_i = D - 2 \cdot r \cdot (1 - \cos(\alpha)) = 3048 \text{ mm} - 2 \cdot 254 \text{ mm} \cdot (1 - 0.866) = 2980 \text{ mm}$$

$$L = \frac{D_i}{2 \cdot \cos(\alpha)} = \frac{2980 \text{ mm}}{2 \cdot 0.866} = 1720 \text{ mm}$$

$$t = \frac{P_0 \cdot L \cdot M}{2 \cdot S \cdot E - 0.2 \cdot P_0} = \frac{19.68 \text{ bar} \cdot 1720 \text{ mm} \cdot 1.401}{2 \cdot 138 \text{ N/mm}^2 \cdot 1 - 0.2 \cdot 19.68 \text{ bar}} = 17.2 \text{ mm}$$

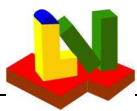
$$P = \frac{2 \cdot S \cdot E \cdot t_0}{L \cdot M + 0.2 \cdot t_0} = \frac{2 \cdot 138 \text{ N/mm}^2 \cdot 1 \cdot 25.4 \text{ mm}}{1720 \text{ mm} \cdot 1.401 + 0.2 \cdot 25.4 \text{ mm}} = 2.903 \text{ MPa}$$

### Equations cone

$$t_1 = \frac{P_0 \cdot D_1}{2 \cdot \cos(\alpha) \cdot (S \cdot E - 0.6 \cdot P_0)} = \frac{19.68 \text{ bar} \cdot 2980 \text{ mm}}{2 \cdot 0.866 \cdot (138 \text{ N/mm}^2 \cdot 1 - 0.6 \cdot 19.68 \text{ bar})} = 24.74 \text{ mm}$$

$$P = \frac{2 \cdot S \cdot E \cdot t_0 \cdot \cos(\alpha)}{D_1 + 1.2 \cdot t_0 \cdot \cos(\alpha)} = \frac{2 \cdot 138 \text{ N/mm}^2 \cdot 1 \cdot 25.4 \text{ mm} \cdot 0.866}{2980 \text{ mm} + 1.2 \cdot 25.4 \text{ mm} \cdot 0.866} = 2.019 \text{ MPa}$$





# ASME BPVC VIII-1 2017

## Example E4.3.7 - E4.3.8 PTB-4-2013

### Appendix: Material documentation

Section 1: Zylinder/E4.3.7 Large End  
Section 1: Kegel/E4.3.7 Large End  
Section 2: Zylinder/E4.3.7 Small End  
Section 2: Kegel/E4.3.7 Small End  
Section 3: Boden/E4.3.8

#### Material specification:

Regulation: ASMET1A:2017Spec. No.: SA-516 Product: Plate  
Material code: K02700-SA-516-70-Class:-Size: Short name: Carbon steel

#### Design conditions and dimensions:

Temperature [°C]: 148,8889 Pressure [bar]: 0  
Thickness [mm]: 46,0375 Outside diameter [mm]: 3901,757

#### Material values for test and design conditions:

	Test condition	Operating condition
Nominal design strength [N/mm²]:	138.00	138.00
Safety factor:		
Allowable stress [N/mm²]:	138.00	138.00
Modulus of elasticity [kN/mm²]:	202,35	195,0667

Wall thickness tolerance [mm]: 0.00 acc. to SA-516

#### Notes:

##### G10 General Requirements

Upon prolonged exposure to temperatures above 425°C, the carbide phase of carbon steel may be converted to graphite. See Nonmandatory Appendix A, A-201 and A-202.

##### S1 Size Requirements

For Section I applications, stress values at temperatures of 450°C and above are permissible but, except for tubular products 75 mm O.D. or less enclosed within the boiler setting, use of these materials at these temperatures is not current practice.

##### T2 Time-Dependent Properties

Allowable stresses for temperatures of 400°C and above are values obtained from time-dependent properties.

--

Creep rupture strength for 100000 h [MPa]:

#### Tensile strength and yield stress at ambient temperature:

Diam./.....	Tensile str....	.....	ReH.....	Rupture.....	Rupture.....
Thick.....	Rm min.....	Rm max.....	.....	elong.....	elong.....
<= mm.....	MPa.....	MPa.....	MPa.....	lengt. %.....	lat. %.....
.....	.....	.....	.....	.....	.....

K-values as function of the temperature

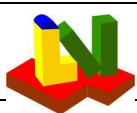
Diam./...	.....	.....	.....	.....	.....	.....	.....	.....
Thickn...	50°C.....	100°C.....	150°C.....	200°C.....	250°C.....	300°C.....	350°C.....	400°C.....
<= mm...	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....
.....	.....	138.....	138.....	138.....	138.....	136.....	128.....	101.....

K-values as function of the temperature

Diam./.....	.....	.....	.....	.....	.....	.....	.....	.....
Thickn....	450°C.....	500°C.....	550°C.....	600°C.....	650°C.....	700°C.....	800°C.....	.....
<= mm.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....
.....	67.1.....	33.6.....	12.9.....	.....	.....	.....	.....	.....

#### Modulus of elasticity in dependence of the temperature:

Static modulus of elasticity in [kN/mm²] at the temperature of



# ASME BPVC VIII-1 2017

## Example E4.3.7 - E4.3.8 PTB-4-2013

-75...	-200...	-125...	25...	100...	150...	200...	250...	300...	350...	400...	450...	500...	550...
209...	216...	212...	202...	198...	195...	192...	189...	185...	179...	171...	162...	151...	137...

### Coefficient of linear expansion:

Thermal coefficient of expansion between 20°C and

Density (20 °C)	100°C..	200°C..	300°C..	400°C..	500°C..	600°C..	700°C..	800°C..	Heat...	Heat...
kg/dm³	10E-6/K	10E-6/K	10E-6/K	10E-6/K	10E-6/K	10E-6/K	10E-6/K	10E-6/K	cond...	capac...
7.85...	12,1...	12,7...	13,3...	13,8...	14,4...	14.8...	15.1...	15.4...	.....	.....